



## Shaft Locking Device Background of the Invention

The invention relates to a shaft locking device for the spindle of a motor-driven, hand-guided work tool ~~according to the preamble of claim 1, in particular.~~

5 A shaft locking device of the aforementioned kind is described in US 3,243,023 which comprises a drive member with four unlocking elements for the locking members of a freewheel and where the unlocking elements are simultaneously designed as cams for engaging a driven member. The torque-transmitting surfaces of the  
10 cams and the driven member are designed to be very small in this solution and they are, therefore, subject to wear.

It is an object of the invention to create a shaft locking device for the spindle of a motor-driven, hand-held work tool that has a long service life.

## Summary of the Invention

15 ~~The object is solved by the characterizing features of claim 1.~~

Due to the spacial separation of unlocking elements and cams, there is space for sufficiently dimensioned torque-transmitting drive surfaces which are practically not subject to any wear.

20 Because of the one-piece design of the spindle and its penetration of the drive member and the driven member, it has a simple design

and a stable support. This support of the spindle in connection with a loosely dimensioned support of the drive member on the spindle contributes to the long service life and the efficiency of the work tool, particularly if the drive member is embodied as a toothed wheel.  
 5 When its teeth are made of plastic, there are noise-related advantages.

Because the axial cams penetrate the catch openings, the entire torque transferring drive surfaces are always active, even in the event of axial mounting deviations of the spindle.

10 Because of the positioning of the unlocking members and of the cams, manufacturing advantages exist for turning and milling the drive member. The cams can axially or radially penetrate or extend into the catch openings.

15 The deep-drawing or extruding of the driven member offers also advantages in regard to manufacturing the cams for the locking members and to flattening of the hub. The driven member can also be embodied as a thick disc which is preferably made of sintered steel. This realizes a particularly low wear and safe operation of the shaft locking device. The number of the locking members depends  
 20 on the magnitude of the operational load.

The fixedly driven closing ring serves to limit the axial movement of the locking members of the free wheel and, thereby, its noise

development. This is particularly important with hammer devices. In the case of different length ratios, the fixedly driven closing ring can be substituted by a standardized disc and, in the event that no hammer device is present, it can even be left out.

5 Advantageously, the cam surfaces of the cams are radially aligned with the drive surfaces of the catch openings. This provides favorable torque transmission and wear conditions.

10 A particularly torsionally strong connection between the driven member and the spindle is achieved when the driven member is positive-lockingly connected to the spindle.

15 Advantageously, the driven member is force-lockingly connected to the spindle. This can be achieved by a press fit, for example. Thereby, a slip-clutch-like slip between the driven member and the spindle is possible on an extreme load acting on the spindle so that a destruction due to breaking-stress can be avoided.

### *Brief Description of the Drawings*

More features can be taken from the further claims. The drawings schematically illustrate the embodiments of the invention which are described in the following. It is shown in:

Fig. 1 a longitudinal section of the shaft locking device,

20 Fig. 2 a cross-section of the shaft locking device,

Fig. 3 a longitudinal section of another shaft locking device,

Fig. 4 a cross-section of another shaft locking device,

Fig. 5 a longitudinal section of a preferred further embodiment of the shaft locking device,

5 Fig. 6 a cross-section of the preferred further embodiment of the shaft locking device.

### Description of Preferred Embodiments

Figs. 1 and 2 show a spindle 1 with a thread 29 for the drill chuck and a ball bearing 2 as well as a drive member 3 and a driven member 4.

10 The interior ring of the ball bearing 2 is axially supported on a shoulder 6 of the spindle 1 by a locking ring 5. A second bearing provided at the driving end of the spindle 1 is not illustrated.

The driving member 3 consists of a toothed wheel 7 that has an end gearing and is supported on the spindle 1. The toothed wheel 7 is axially fixed with axial play by a further locking ring 8 and a shoulder 9 of the spindle 1. For noise-related reasons, the gearing can be comprised of plastic. At the end face of the toothed wheel 7 facing the driven member 4, three unlocking members 10 and three axial cams 11 are arranged in pairs concentrically with respect to one another. The three pairs each have a common center line 12, and

neighboring center lines 12 are provided at equal spacings.

The driving member 3 can also be embodied as a planetary gearing and be provided with a two-position gear box.

5 The driven member 4 is provided with a deep-drawn, pot-shaped sheet metal member 13, whose hub 14 is fixedly connected to the spindle 1 by two flattened portions 15. The hub 14 can also be fixedly connected to the spindle 1 by a press fit or other kind of frictional connection or by positive locking. In the wall 16 of the sheet metal member 13 facing the drive member 3, three catch  
10 openings 30 embodied as ring segments 17 are provided which are radially engaged with rotary play by the three axial cams 11 of the toothed wheel 7. The cam surfaces 18 and drive surfaces 19 of the sheet metal member 13, which face one another in the direction of rotation, are radially aligned and are positioned opposite one  
15 another with a clearance.

The sheet metal member 13 is provided with an outer ring-shaped wall 20 having an outer shape that is embodied as cams 21 for the locking members 22. The locking members 22 are positioned as  
20 pairs in the area between the catch openings 30 and are arranged between the outer wall 20 and a ring 23 fixedly attached to the housing. The locking members 22 are being pressed apart by spring elements 24 (for example, metal or plastic springs) and are held in a clamping position on the cams 21. The cams 21, the locking

members 22, and the ring 23 attached to the housing together form a freewheel 28. The number of the locking member pairs is dependent on the operational torque to be expected.

5 The unlocking members 10 embrace the outer, ring-shaped wall 20 of the metal member 13 at a spacing. They are provided at their end faces 25 facing the locking members 22 a shaped surface matching the locking members 22. The distance between the shaped surface and the locking members 22 is smaller than the one between the cam surfaces 18 and the drive surfaces 19.

10 The ring 23, fixedly attached to the non-represented housing of the work tool, is positive-lockingly connected, for example, by a toothed ring 26.

15 The sheet metal member 13 is provided at the side facing away from the driven member 4 with a closing member 27 that is fixedly connected to the spindle 1 and limits the axial movement of the locking members 22 and, thereby, the resulting noise development. This is particularly important with hammer devices. If this is missing, the closing member 27 can be eliminated.

20 Figures 3 and 4 illustrate another version of the shaft locking device. The same parts carry the same designation as in Figs. 1 and 2, analogous parts carry a designation extended by an "a".

5 In this version, differently designed cams 11a of a differently designed toothed wheel 7a of a differently designed drive member 3a engage with play corresponding radial recesses 17a of an extruded member 13a belonging to a differently designed driven member 4a. This extruded member 13a is fixedly connected to a different spindle 1a.

10 At the circumference of the extruded member 13a, between the radial recesses 17a, cams 21 (according to Figs. 1 and 2) are provided that can be connected by locking members 22 to the ring 23 fixedly attached to the housing. The locking members 22 are held axially by a closing member 27a that is connected to the differently designed cams 11a.

15 The torque of the drive motor is transmitted by differently designed cam surfaces 18a to differently designed drive surfaces 19a of the extruded member 13a.

20 Differently designed unlocking members 10a embrace the extruded member 13a at a spacing. The distance between the differently designed end surfaces 25a of the differently designed unlocking members 10a facing the locking members 22 and the locking members 22 is smaller than between the differently designed cam surfaces 18a and the differently designed drive surfaces 19a of the extruded member 13a.

Figures 5 and 6 illustrate another preferred embodiment of the shaft clamping device. The same parts carry the same designation as in Figs. 1 and 2, analogous parts carry a designation extended by a "b".

5 In this embodiment, the cams 11b of a different toothed wheel 7b are similarly designed as the cams 11a shown in Figs. 4, whereby differently designed cam surfaces 18b of the cams 11b are approximately radially aligned with different drive surfaces 19b of the catch opening 30. These are embodied as differently designed  
10 radial recesses 17b. The cams 11b project radially and axially with a play into the corresponding recesses 17b of a differently designed driven member 4b which is embodied as a thick disc 13b. The driven member 4b is provided with a different hub 14b which is fixedly and force-lockingly connected to a different spindle 1b by  
15 press fit.

At the circumference of the disc 13b dovetailed cams 21 according to Figs. 1 to 4 are provided between the radial recesses 17b. The cams 21 can be connected to the ring 23, fixedly attached to the housing, by locking members 22. The locking members 22 are held  
20 axially by a differently designed closing member 27b which is fixedly connected to the spindle 1b.

The torque of the drive motor is transmitted by different cam surfaces 18b to different drive surfaces 19b of the disc 13b.



The illustrated shaft locking devices operate as follows:

The illustrated shaft locking devices operate as follows:

On operation of the motor, the toothed wheel 7, 7a, 7b is rotated on the spindle 1, 1a, 1b. Thereby, first, the first locking elements 22 as viewed in the direction of rotation are shifted into the recess of the cams 21 by the unlocking members 10, 10a, 10b and are thereby unlocked. Only then, the cam surfaces 18, 18a, 18b engage the drive surfaces 19, 19a, 19b of the driven member 4, 4a, 4b whereby also the second locking member 22 and, thereby, the freewheel 28 as an entity are unlocked. The spindle 1, 1a, 1b, together with the drive member 3, 3a, 3b and the driven member 4, 4a, 4b, can now freely rotate and transmit torque.

20 When the drive of the spindle 1, 1a, 1b rests, the axial cams 11, 11a, 11b and the unlocking members 10, 10a, 10b are positioned in neutral position without any contact to the driven member 4, 4a, 4b and the locking members 22. The latter are held in a locking

position by the spring elements 24. On rotating the spindle 1, 1a, 1b by hand, one of the locking members 22 is engaged in the direction of the locking position and the other one in the unlocking direction, respectively, whereby the locking members 22 switch their function when the direction of rotation is changed. In this manner, a firm rotation connection to the housing of the work tool is always present when the spindle 1, 1a, 1b is rotated manually. This situation can be utilized for opening or closing, or screwing on and off the drill chuck with one hand. Thereby, a double bush drill chuck is not required which is expensive and has to be handled with both hands. Another advantage of the shaft locking device is that a screw can be manually screwed in and out by using the work tool as an effective screw driver.

Ins  
a, >